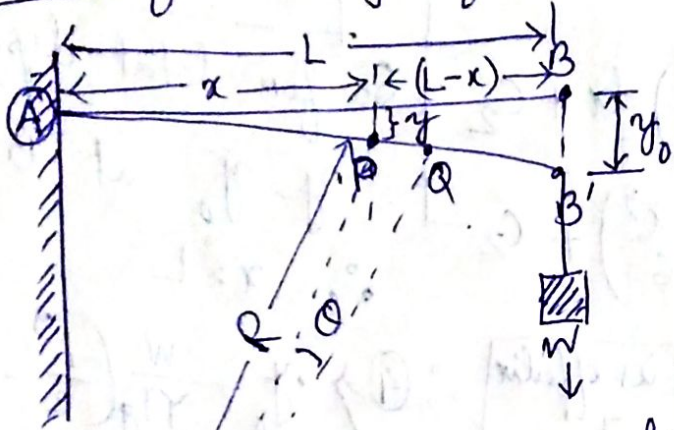


Expression for depression produced in a cantilever:

A cantilever is a beam fixed horizontally to support heavy loads.

Case (a). If the weight of the cantilever is negligible



Let us consider AB be the neutral axis of a cantilever of length L. It is fixed at A and loaded at B by a load W. As a result, the point B shifts B' through a vertical distance  $y_0$  (total depression). Now, let consider any

arbitrary section P of the cantilever at a distance  $x$  from the fixed point A.

Moment of the force at P =  $W(L-x)$   
 But bending moment of a beam (cantilever) =  $\frac{Y I_g}{R}$

$$W(L-x) = \frac{Y I_g}{R} \Rightarrow \frac{1}{R} = \frac{W}{Y I_g} (L-x) \quad \text{--- (1)}$$

From geometry we know,  $\frac{1}{R} = \frac{d^2 y / dx^2}{\left[1 + \left(\frac{dy}{dx}\right)^2\right]^{3/2}}$ ; But here slope = 0 = very small  
 as bending is very small  $\therefore \left(\frac{dy}{dx}\right) = 0$

$$\therefore \text{(2)} \Rightarrow \frac{1}{R} = \frac{d^2 y / dx^2}{1} = \frac{d^2 y}{dx^2}$$

Using this in (1), we have

$$\begin{aligned} \frac{d^2 y}{dx^2} &= \frac{W}{Y I_g} (L-x) \\ \Rightarrow \frac{d\left(\frac{dy}{dx}\right)}{dx} &= \frac{W}{Y I_g} (L-x) \\ \Rightarrow d\left(\frac{dy}{dx}\right) &= \frac{W}{Y I_g} (L-x) dx \end{aligned}$$

Integrating both sides

$$\int d\left(\frac{dy}{dx}\right) = \frac{W}{Y I_g} \int (L-x) dx + C_1$$

$$\left(\frac{dy}{dx}\right) = \frac{W}{Y I_g} \left(Lx - \frac{x^2}{2}\right) + C_1 \quad \text{--- (3)}$$

At  $x=0, \frac{dy}{dx} = 0, C_1 = 0$

$\therefore \textcircled{3} \Rightarrow \left(\frac{dy}{dx}\right) = \frac{W}{\gamma I_g} \left(Lx - \frac{x^2}{2}\right)$

$\Rightarrow dy = \frac{W}{\gamma I_g} \left(Lx - \frac{x^2}{2}\right) dx$

Again integrating both sides

$\int dy = \int \frac{W}{\gamma I_g} \left(Lx - \frac{x^2}{2}\right) dx + C_2$

$\Rightarrow y = \frac{W}{\gamma I_g} \left(L \frac{x^2}{2} - \frac{x^3}{6}\right) + C_2$

at  $x=0, y=0$

$\therefore C_2 = 0$

$\therefore y = \frac{W}{\gamma I_g} \left(L \frac{x^2}{2} - \frac{x^3}{6}\right)$

So for total depression,

$y = y_0$  at B.

$\therefore x = L$

$\therefore \textcircled{4} \Rightarrow y_0 = \frac{W}{\gamma I_g} \left(\frac{L^3}{2} - \frac{L^3}{6}\right)$

$\Rightarrow y_0 = \frac{WL^3}{3\gamma I_g} \rightarrow \textcircled{5}$

Case b: When weight of the cantilever is effective  
[Description same as case a]

Let  $w$  be the weight per unit length of the cantilever, and this weight also produces bending in it, in addition to  $W$ .

$\therefore$  Moment of force at P is  $= W(L-x) + w(L-x) \frac{(L-x)}{2}$   
 $= W(L-x) + \frac{w(L-x)^2}{2}$

$\therefore W(L-x) + \frac{w(L-x)^2}{2} = \frac{\gamma I_g}{R} \rightarrow \textcircled{6}$

$\Rightarrow \frac{1}{R} = \frac{W}{\gamma I_g} (L-x) + \frac{w}{\gamma I_g} \frac{(L-x)^2}{2}$

$\Rightarrow \frac{d^2y}{dx^2} = \frac{W}{\gamma I_g} (L-x) + \frac{w}{\gamma I_g} \frac{(L-x)}{2}$

Integrate both sides

$\int d\left(\frac{dy}{dx}\right) = \frac{W}{\gamma I_g} \int (L-x) dx + \frac{w}{2\gamma I_g} \int (L-x)^2 dx + C_1$

$\Rightarrow \left(\frac{dy}{dx}\right) = \frac{W}{\gamma I_g} \left(Lx - \frac{x^2}{2}\right) + \frac{w}{2\gamma I_g} \left[Lx + \frac{x^3}{3} - 2L\frac{x^2}{2}\right] + C_1$

$$\rightarrow \left(\frac{dy}{dx}\right) = \frac{w}{4I_g} \left(Lx - \frac{x^2}{2}\right) + \frac{w}{27I_g} \left[Lx^2 + \frac{x^3}{3} - Lx^2\right] + C_1$$

at  $x=0$ ,  $\frac{dy}{dx} = 0$ ,  $C_1 = 0$

$$\therefore \left(\frac{dy}{dx}\right) = \frac{w}{4I_g} \left(Lx - \frac{x^2}{2}\right) + \frac{w}{27I_g} \left(Lx + \frac{x^3}{3} - Lx^2\right)$$

Again integrating both sides

$$\int dy = \frac{w}{4I_g} \int \left(Lx - \frac{x^2}{2}\right) dx + \frac{w}{27I_g} \int \left(Lx + \frac{x^3}{3} - Lx^2\right) dx + C_2$$

$$\Rightarrow y = \frac{w}{4I_g} \left(L\frac{x^2}{2} - \frac{x^3}{6}\right) + \frac{w}{27I_g} \left(L\frac{x^2}{2} + \frac{x^4}{12} - L\frac{x^3}{3}\right) + C_2$$

At  $x=0$ ,  $y=0$ ,  $C_2=0$

$$\therefore y = \frac{w}{4I_g} \left(L\frac{x^2}{2} - \frac{x^3}{6}\right) + \frac{w}{27I_g} \left(L\frac{x^2}{2} + \frac{x^4}{12} - L\frac{x^3}{3}\right) \rightarrow \textcircled{7}$$

Now total depression at B,  $y = y_0$ ,  $x = L$

$$\therefore \textcircled{6} \Rightarrow y_0 = \frac{w}{4I_g} \frac{L^3}{3} + \frac{w}{27I_g} \left(\frac{L^4}{2} + \frac{L^4}{12} - \frac{L^4}{3}\right)$$

$$= \frac{w}{4I_g} \frac{L^3}{3} + \frac{w}{27I_g} \left(\frac{L^4}{4}\right)$$

$$= \frac{w}{4I_g} \frac{L^3}{3} + \frac{w}{72I_g} \frac{L^4}{L}$$

$$= \frac{wL^3}{36I_g} + \frac{(wL)L^3}{87I_g}$$

$$= \frac{wL^3}{36I_g} + \frac{w_1 L^3}{87I_g}$$

$w_1 = wL$   
is the total weight of the beam

$$y_0 = \left(w + \frac{3}{8}w_1\right) \frac{L^3}{36I_g} \rightarrow \textcircled{8}$$

Case c When no external weight is applied and only the weight of the beam itself is producing bending.

[ Description's same as case (a) ]  
but  $W=0$

$\therefore$  If  $w$  is the weight of the cantilever per unit length,

then (6) (8)  $\rightarrow w(L-x) \frac{(L-x)}{2} = \frac{Y I_g}{R} \rightarrow$  (9)

$\rightarrow \frac{1}{R} = \frac{w}{2Y I_g} (L-x)$

$\rightarrow \frac{d^2 y}{dx^2} = \frac{w}{2Y I_g} (L-x)^2$

Integrating twice (write the procedure as case (b))

$$y_0 = \frac{wL^3}{8Y I_g}; \quad w_1 = wL \rightarrow$$
 (10)

Case d Depression produced in a cantilever, fixed at both ends and loaded in the middle

Therefore, for this case

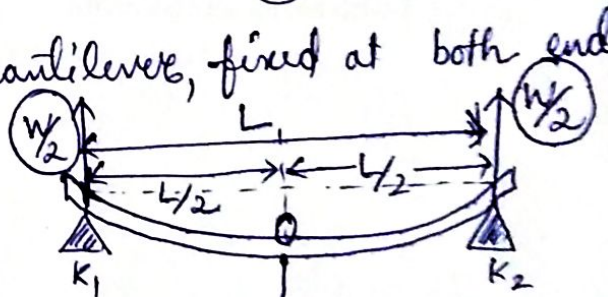
$w \rightarrow \frac{W}{2}, \quad L \rightarrow \frac{L}{2}$

$\therefore$  Total depression (from case (a))

$$y_0 = \frac{wL^3}{3Y I_g}$$

$$\rightarrow y_0 = \frac{(\frac{W}{2})(\frac{L}{2})^3}{3Y I_g}$$

$$\rightarrow y_0 = \frac{WL^3}{48Y I_g}$$



It is the combination of two beams fixed at one point and loaded at the other. [Repeat the case (a)]

